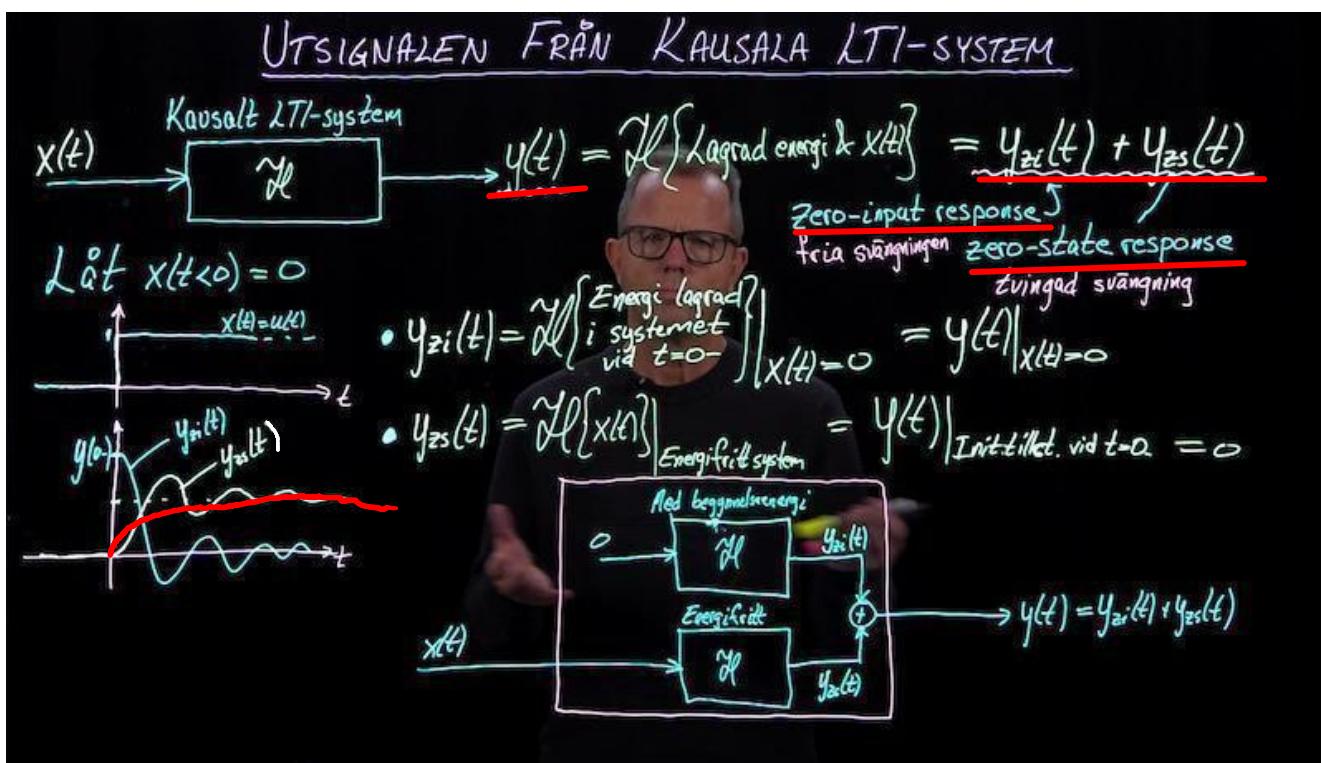


Linjära System – Föreläsning 9: Tidsdomänanalys av LTI-system

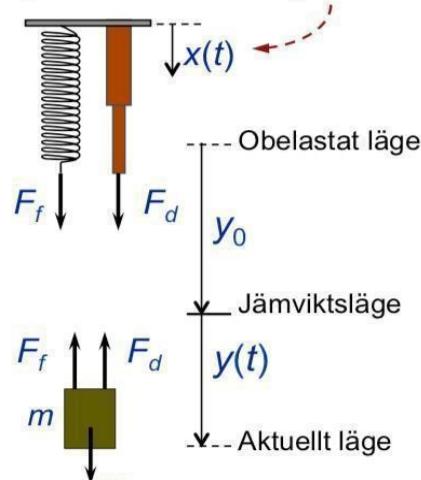
VIDEO 1



Systemexempel 1 – Mekaniskt svängningssystem, massa i dämpad fjäder

Svängande dämpad fjäder – frilägg och sätt ut krafter:

Insignal: ändrad infästningspunkt



$$\text{Fjäderkraften } F_f = k \cdot y_{tot}(t) = k \cdot (y_0 + y(t) - x(t))$$

$$\text{Dämpkraften } F_d = c \cdot (y_{tot}(t))' = c \cdot (y'(t) - x'(t))$$

$$\text{Tyngdkraften } F_m = m \cdot g \quad (g = \text{tyngdaccelerationen})$$

$$\text{Newtons 2:a lag: } F_m - F_f - F_d = m \cdot y''(t)$$

$$\Rightarrow m \cdot y''(t) + c \cdot y'(t) + k \cdot y(t) = m \cdot g - k \cdot y_0 + c \cdot x'(t) + k \cdot x(t)$$

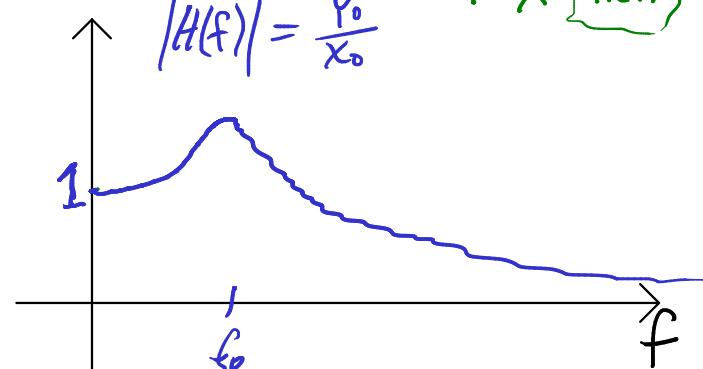
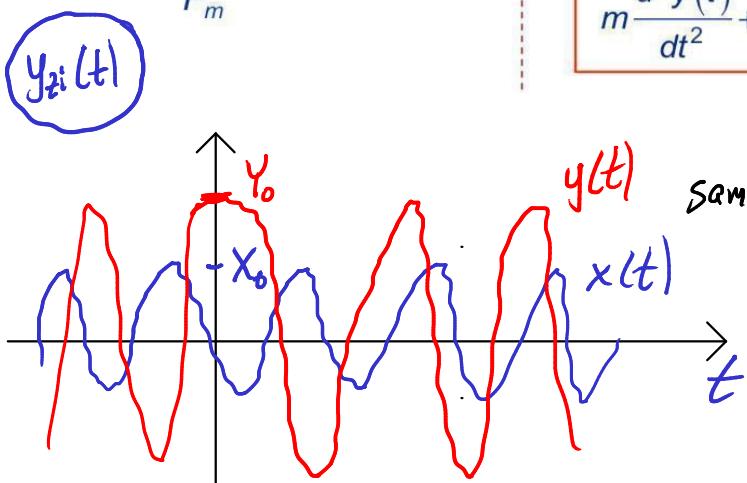
$$\text{Vid vila är } x=0, x'=0, y=0, y'=0, y''=0 \Rightarrow m \cdot g = k \cdot y_0$$

\Rightarrow

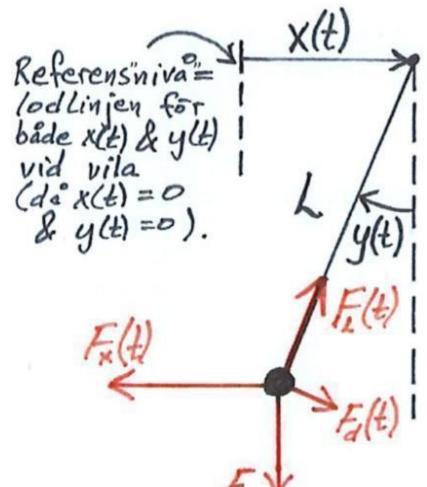
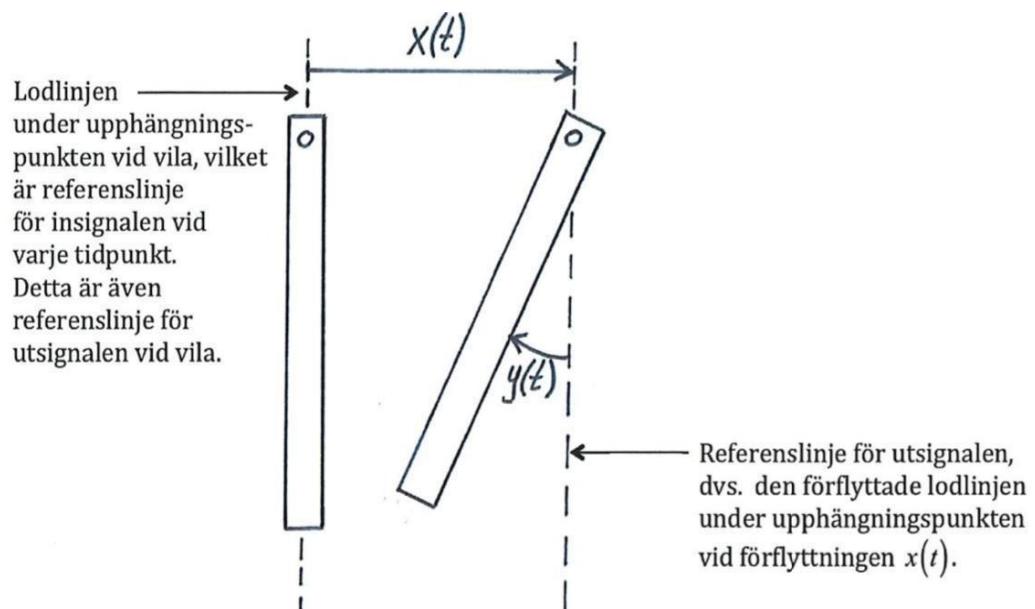
$$m \frac{d^2y(t)}{dt^2} + c \frac{dy(t)}{dt} + k \cdot y(t) = c \frac{dx(t)}{dt} + k \cdot x(t)$$

beror på $\omega = \sqrt{\frac{k}{m}}$

$$Y=X \cdot H(f)$$



Systemexempel 2 – Mekaniskt svängningssystem, pendlande linjal



Ideal modell

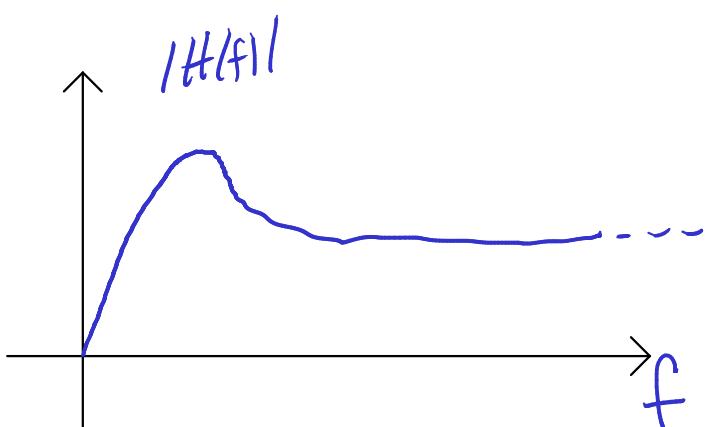
$$\text{Newtons 2:a lag} \Rightarrow y'' + \frac{c}{m}y' + \frac{g}{m}\sin(y) = \frac{\cos(y)}{L} \cdot x''$$

Icke-linjärt system, p.g.a. $\sin(y)$ och $\cos(y)$

\Rightarrow Linjärisera, dvs. approximera med linjär modell:

Om vinkeln y är liten $\Rightarrow \sin(y) \approx y, \cos(y) \approx 1$

$$\Rightarrow \boxed{\frac{d^2y(t)}{dt^2} + \frac{c}{m} \frac{dy(t)}{dt} + \frac{g}{L} y(t) = \frac{1}{L} \frac{d^2x(t)}{dt^2}}$$



DEN FRIA SVÄNGNINGEN, ZERO-INPUT RESPONSE $y_{zi}(t)$

$x(t)=0$



Differentialekvationsbeskrivning:

$$a_N \cdot \frac{d^N y(t)}{dt^N} + a_{N-1} \cdot \frac{d^{N-1} y(t)}{dt^{N-1}} + \dots + a_1 \cdot \frac{dy(t)}{dt} + a_0 \cdot y(t) = b_M \cdot \frac{d^M x(t)}{dt^M} + b_{M-1} \cdot \frac{d^{M-1} x(t)}{dt^{M-1}} + \dots + b_1 \cdot \frac{dx(t)}{dt} + b_0 \cdot x(t)$$

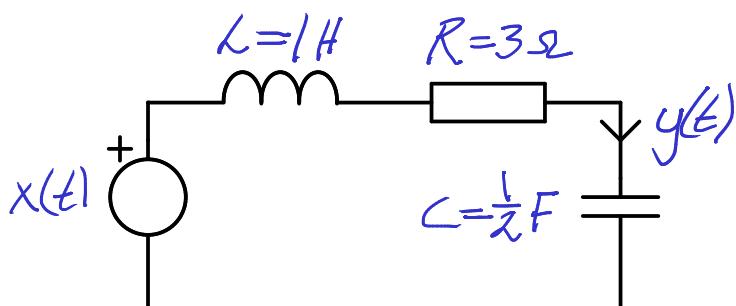
(Vanligt: $N > M$)

Deriveringsoperatorn \mathcal{D} : $\mathcal{D}^i y(t) = \frac{d^i y(t)}{dt^i}$

$$\Rightarrow Q(\mathcal{D})y(t) = P(\mathcal{D})x(t) \quad \text{där} \quad \begin{cases} Q(\mathcal{D}) = a_N D^N + a_{N-1} D^{N-1} + \dots + a_1 D + a_0 \\ P(\mathcal{D}) = b_M D^M + b_{M-1} D^{M-1} + \dots + b_1 D + b_0 \end{cases}$$

$$\Rightarrow \text{lös } Q(\mathcal{D})y_{zi}(t) = 0 \quad \Rightarrow \quad y_{zi}(t) = \sum \text{(karaktäristiska termer)}$$

$e^{\lambda t}, t^n e^{\lambda t}, e^{\alpha t} \cos(\beta t)$

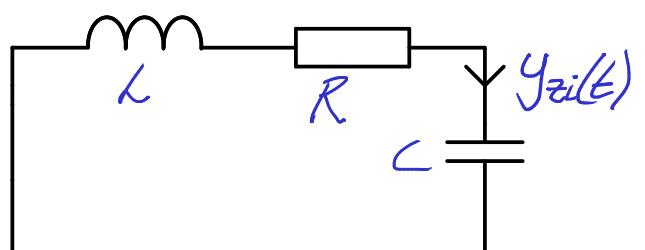
Exempel:

$$(D^2 + 3D + 2)y(t) = D x(t)$$

$$\frac{d^2 y(t)}{dt^2} + 3 \frac{dy(t)}{dt} + 2y(t) = \frac{dx(t)}{dt}$$

$$y(0) = 0, \quad y'(0) = -5$$

I VT2 kommer ni, i En- och flervariabelkursen, att lära er hur man löser (dvs. beräknar $y(t)$) sådana här differentialekvationer!

Beräkna $y_{zi}(t)$ Ekvivalent krets då $x(t)=0$:

$$(D^2 + 3D + 2)y_{zi}(t) = 0$$

$$\lambda^2 + 3\lambda + 2 = 0 \quad \text{Karakteristiska ekvationen}$$

$$\Rightarrow \lambda_1 = -1, \quad \lambda_2 = -2 \quad \Rightarrow \underline{y_{zi}(t) = K_1 \cdot e^{\lambda_1 t} + K_2 \cdot e^{\lambda_2 t}}$$

$$h(t) = A \cdot e^{\lambda_1 t} + B \cdot e^{\lambda_2 t}$$

Systemets ordning = 2

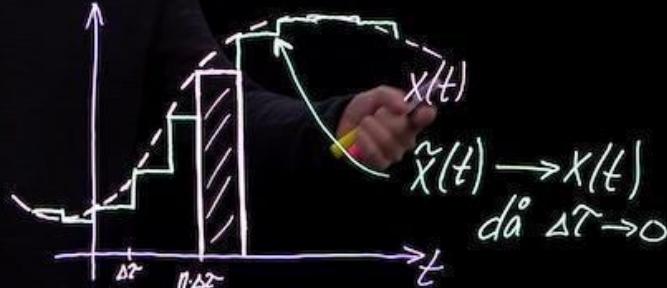
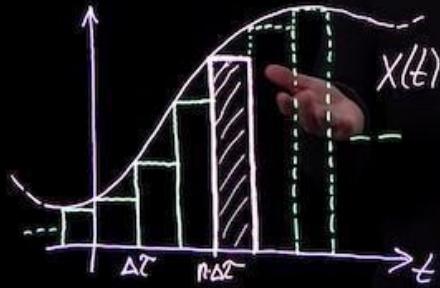
DEN Tvingade Svängningen, ZERO-STATE RESPONSE $y_{zs}(t)$

Energifritt LTI-system

$$X(t) \xrightarrow{\text{LTI}} Y(t) = Y_{zi}(t) + Y_{zs}(t) = \begin{cases} Y_{zi}(t) = 0 & \text{följ energifritt} \\ Y_{zs}(t) = \mathcal{Y}\{X(t)\} \end{cases}$$

Låt $X(t) = a_1 \cdot X_1(t) + a_2 \cdot X_2(t) + \dots = \sum_n a_n \cdot X_n(t)$

Linjärt system $\Rightarrow Y(t) = a_1 \cdot Y_1(t) + a_2 \cdot Y_2(t) + \dots = \sum_n a_n \cdot Y_n(t)$

DEN Tvingade Svängningen, ZERO-STATE RESPONSE $y_{zs}(t)$

$P(t) = u(t) - u(t-\Delta t)$

$$X(t) \xrightarrow{\text{Energifritt LTI}} Y(t) = \sum_n a_n \cdot Y_n(t)$$

$X(n\Delta t)$

$P(t) = \sum_n a_n \cdot P(t-n\Delta t)$

$\hat{Y}(t) = \sum_n a_n \cdot Y_n(t)$

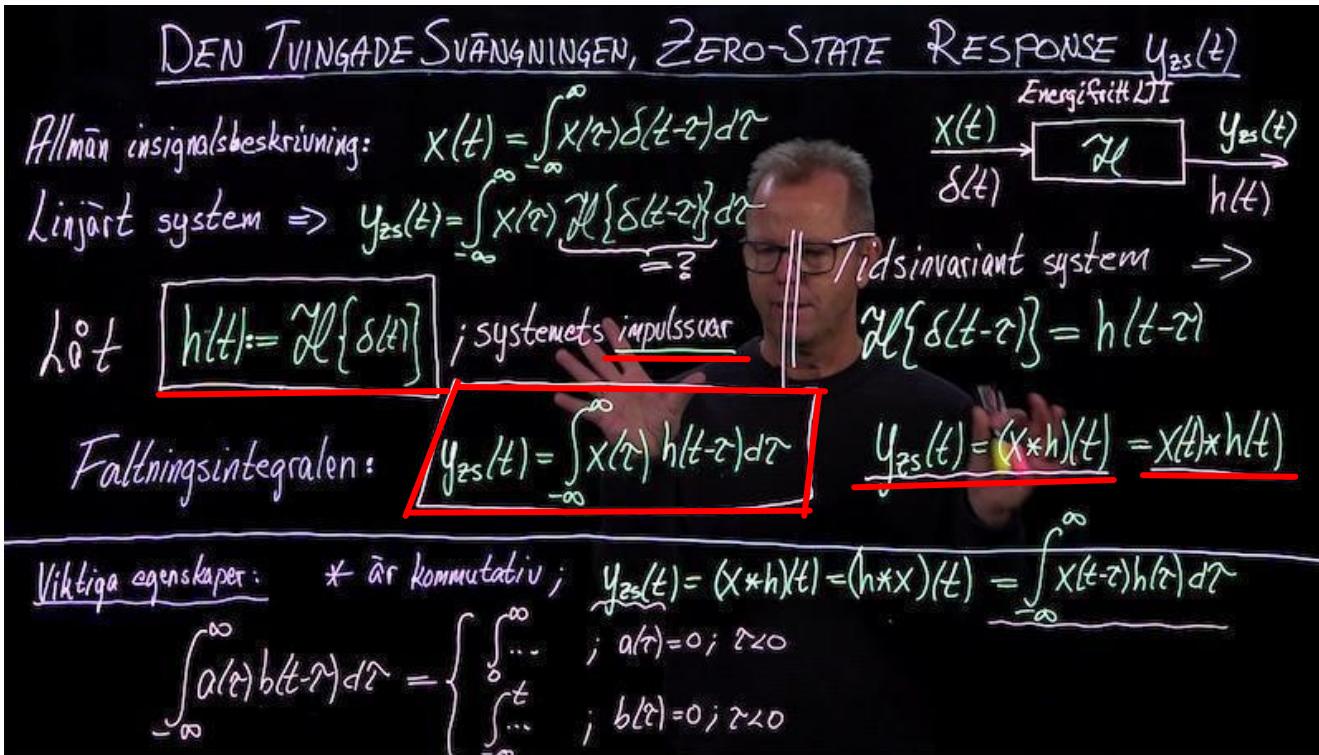
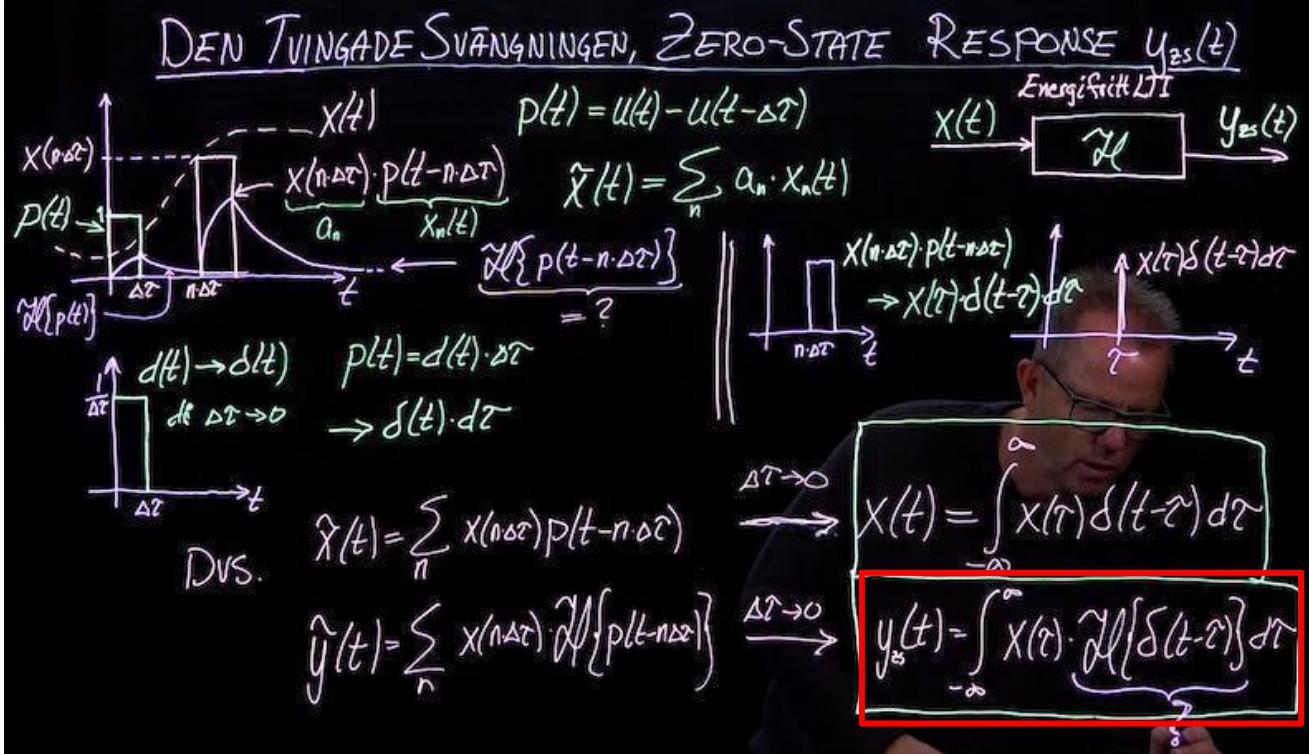
$\hat{Y}(t) = \sum_n a_n \cdot X(n\Delta t) \cdot P(t-n\Delta t)$

$\hat{Y}(t) = \sum_n a_n \cdot X(n\Delta t) \cdot \mathcal{Y}\{P(t-n\Delta t)\}$

$\hat{Y}(t) = \sum_n a_n \cdot Y_n(t)$

$\hat{Y}(t) = \sum_n a_n \cdot Y_n(t) = \sum_n X(n\Delta t) \cdot \mathcal{Y}\{P(t-n\Delta t)\} \xrightarrow[\Delta t \rightarrow 0]{d\delta} y_{zs}(t)$

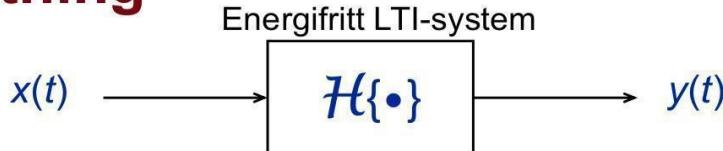
Linjärt system $\Rightarrow \hat{Y}(t) = \sum_n a_n \cdot Y_n(t) = \sum_n X(n\Delta t) \cdot \mathcal{Y}\{P(t-n\Delta t)\} \xrightarrow[\Delta t \rightarrow 0]{d\delta} y_{zs}(t)$



OBS: Kursboken använder λ i stället för τ som integrationsvariabel

Faltning

VIDEO 4.1



$$\mathcal{H} = \text{systemoperatorn}; \quad y(t) = \mathcal{H}\{x(t)\}$$

♦ Från def. av $\delta(t)$ följer: $x(t) = \int_{-\infty}^{\infty} x(\tau) \delta(t - \tau) d\tau$

$$\Rightarrow \underline{y(t)} = \mathcal{H}\{x(t)\} = \mathcal{H}\left\{ \int_{-\infty}^{\infty} x(\tau) \delta(t - \tau) d\tau \right\} = / \text{Linjärt} /$$

$$= \int_{-\infty}^{\infty} x(\tau) \mathcal{H}\{\delta(t - \tau)\} d\tau = \begin{cases} \text{Tids-} \\ \text{invariant} \end{cases} = \int_{-\infty}^{\infty} x(\tau) h(t - \tau) d\tau$$

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Faltningsintegralerna konvergerar garanterat om

$$\int_{-\infty}^{\infty} |x(t)| dt < \infty \text{ och } |h(t)| < \infty \quad \text{eller} \quad |x(t)| < \infty \text{ och } \int_{-\infty}^{\infty} |h(t)| dt < \infty$$

VIDEO 4.2

$X(t)$ $\rightarrow \boxed{\text{LTI}}$ $y(t) = \mathcal{H}\{x(t)\}$
 $X_1(t)$ $y_1(t)$
 $X_2(t)$ $y_2(t)$
 $a_1 X_1(t) + a_2 \cdot X_2(t)$ / Linjärt / $a_1 \cdot y_1(t) + a_2 \cdot y_2(t)$

$$\delta(t) \qquad h(t)$$

$$\delta(t-z) \quad / \text{Tidsinvariant} / \quad h(t-z)$$

$$x(z) \cdot \delta(t-z)$$

$$x(t) = \int_{-\infty}^{\infty} x(z) \delta(t-z) dz \quad / \text{LTI} /$$

$$y(t) = \int_{-\infty}^{\infty} x(z) h(t-z) dz$$

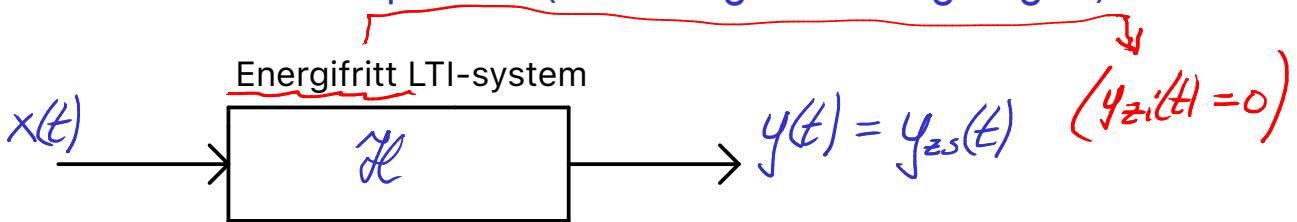
$$= (x * h)(t)$$

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Slutsats efter video 3 & 4 ovan:

$$y(t) = y_{zi}(t) + y_{zs}(t)$$

Utsignalens zero-statekomponent (den tvingade svängningen):



där

$$y_{zs}(t) = (x * h)(t) = \int_{-\infty}^{\infty} x(\tau) h(t-\tau) d\tau = \int_{-\infty}^{\infty} x(t-\tau) h(\tau) d\tau$$

Eller $x(t) * h(t)$

Faltningsintegralen/-erna

där $h(t) = \text{impulssvar}$

Räkneexempel – faltning

Ett visst energifritt LTI-system har impulssvaret $h(t) = 6e^{-2t}u(t)$.

a) Beräkna systemets utsignal $y(t)$ då dess insignal är

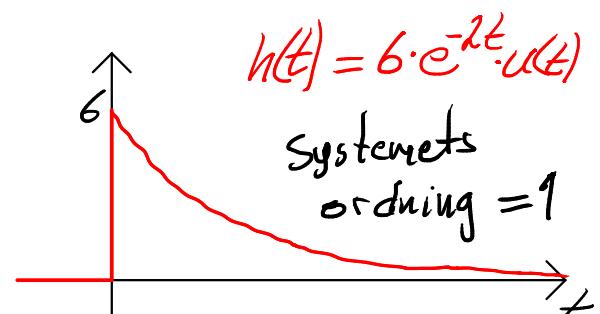
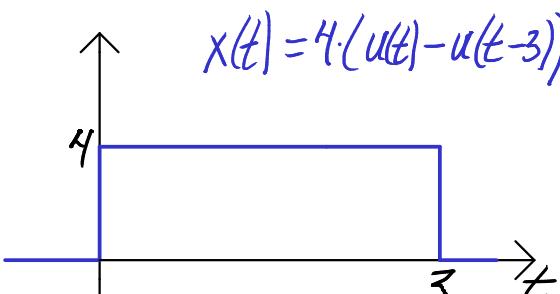
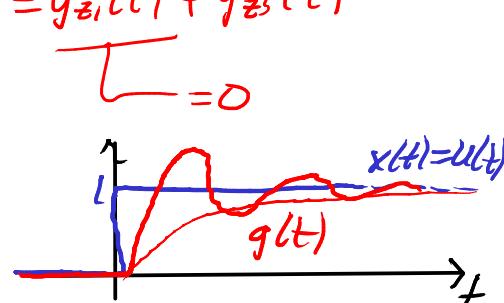
$$x(t) = 4(u(t) - u(t-3)) \quad \boxed{=} = y_{zi}(t) + y_{zs}(t)$$

b) Beräkna systemets stegsvar $g(t)$

$$= y(t) \text{ då } x(t) = u(t)$$

c) Bestäm systemets kausalitetsegenskap

d) Bestäm systemets stabilitetsegenskap



a) Beräkna systemets utsignal $y(t)$

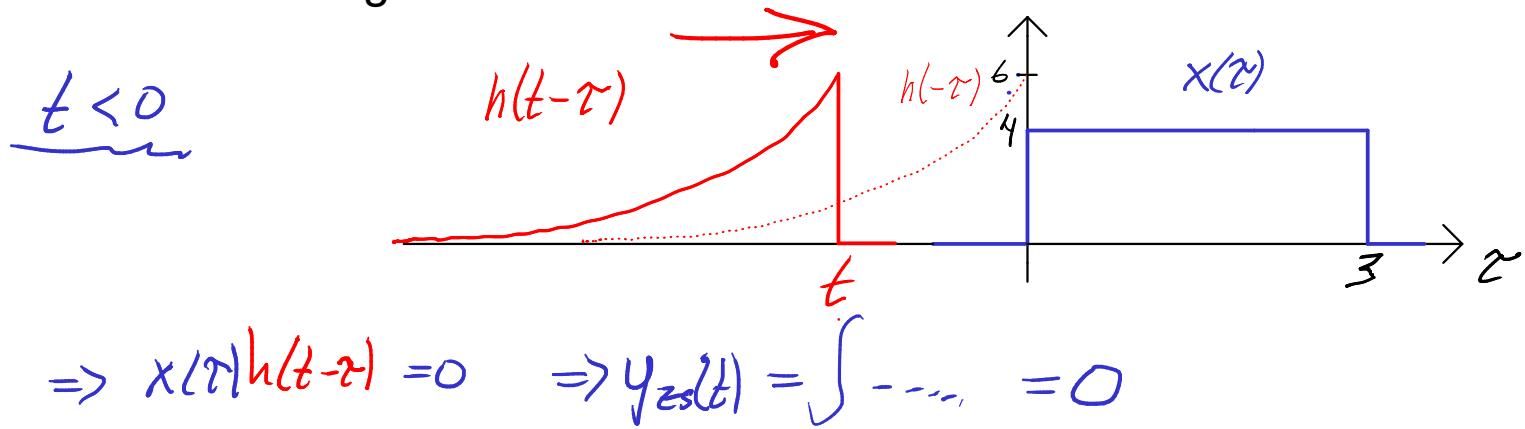
$$y(t) = y_{zs}(t) \quad \text{ty energifritt system}$$

$$u(t-\tau)^2 \cdot \underline{\underline{h(-\tau)}}$$

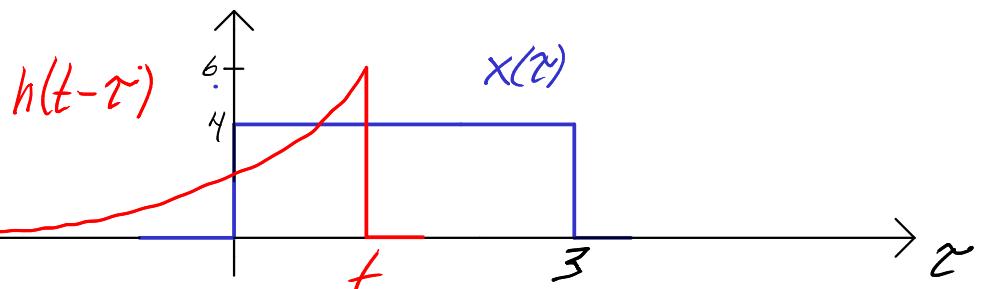
$$\underline{\underline{h(t-\tau)}} = h(-\tau + t)$$

$$y_{zs}(t) = (x * h)(t) = \int_{-\infty}^{\infty} x(\tau) \underline{\underline{h(t-\tau)}} d\tau$$

"Grafisk faltning":

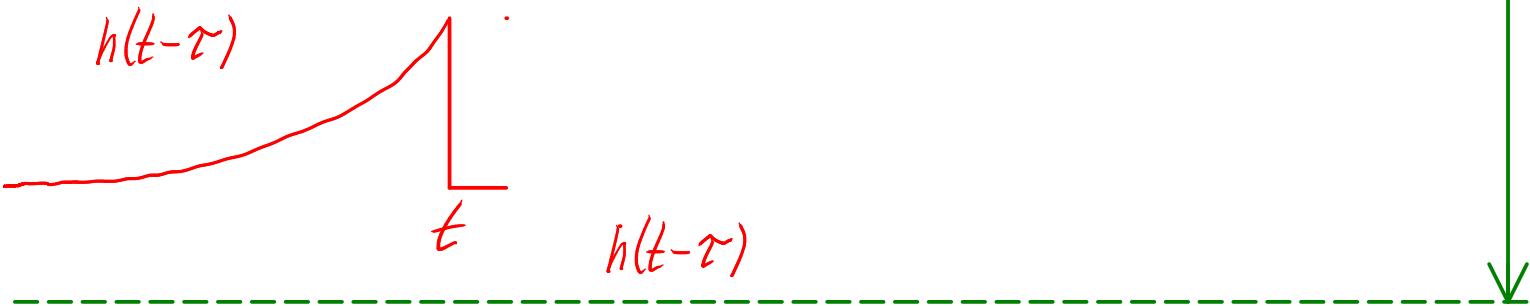


$$\begin{aligned} t > 0 \\ t < 3 \\ \text{dvs. } \underline{\underline{0 \leq t < 3}} \end{aligned}$$

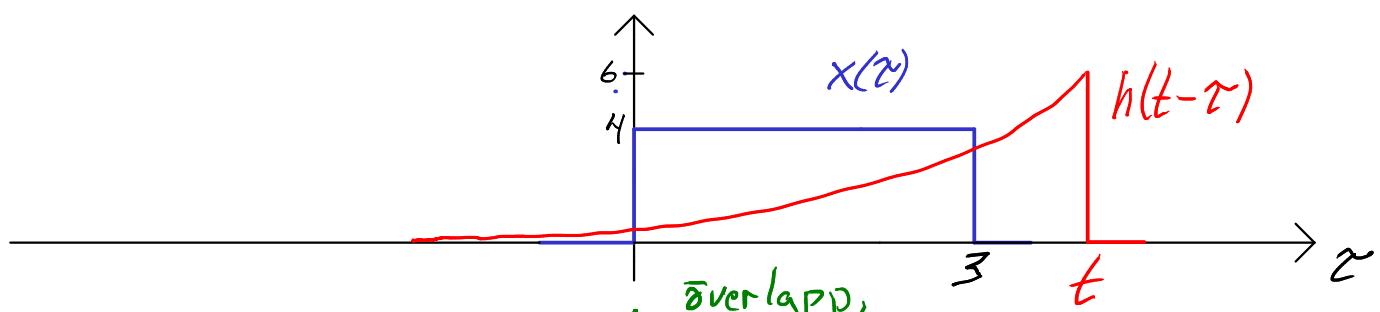


$$\begin{aligned} y_{zs}(t) &= \int_{-\infty}^0 0 \cdot h(t-\tau) d\tau + \int_0^t 4 \cdot 6 \cdot e^{-2(t-\tau)} d\tau + \int_t^{\infty} 0 \cdot h(t-\tau) d\tau \\ &= 24e^{-2t} \int_0^t e^{2\tau} d\tau = \dots = \underline{\underline{12(1 - e^{-2t})}} \end{aligned}$$

$\overbrace{\hspace{10em}}$ överlapp, bide x & $h \neq 0$



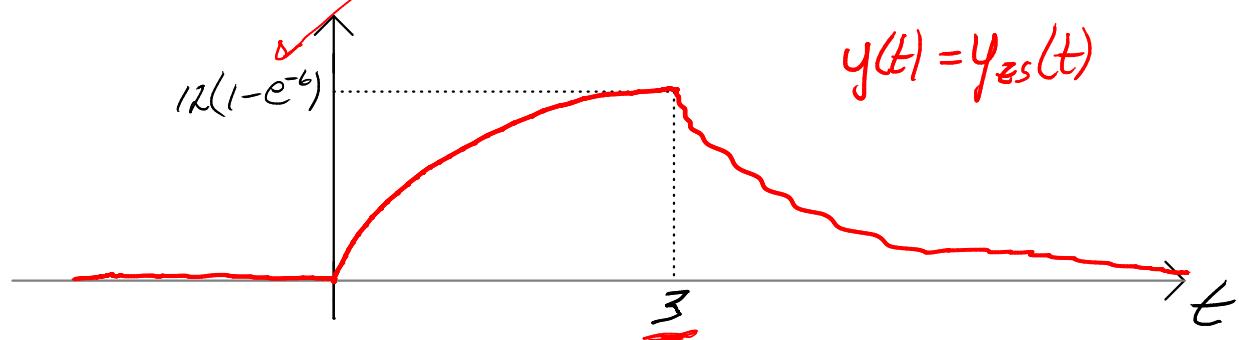
$t \geq 3$



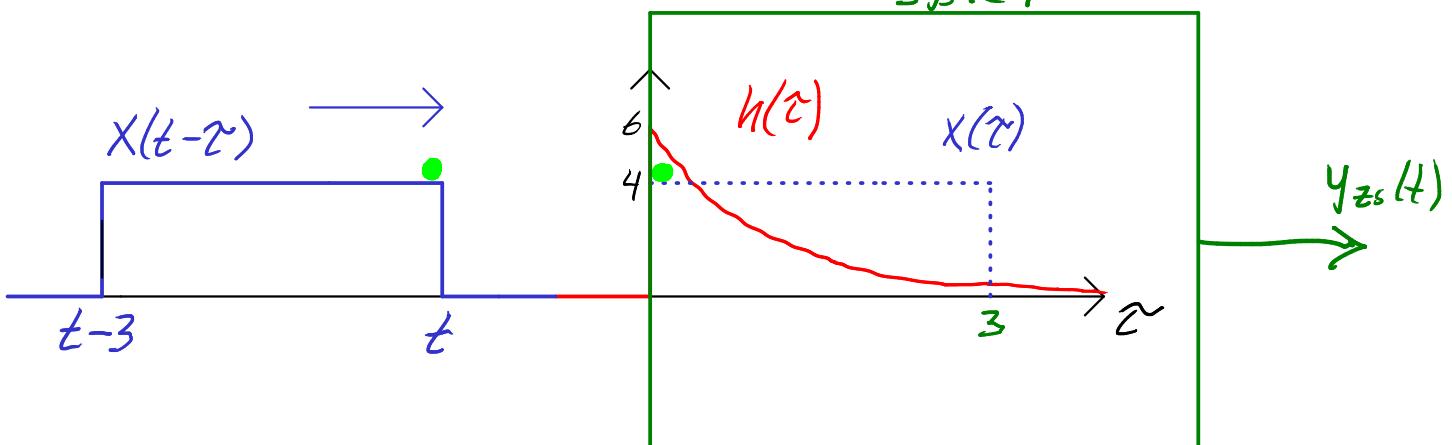
$$y_{zs}(t) = \int_0^3 4 \cdot 6 e^{-2(t-\zeta)} d\zeta = 24 e^{-2t} \int_0^3 e^{2\zeta} d\zeta = \dots = 12(e^6 - 1) e^{-2t}$$

$$\therefore y_{zs}(t) = \begin{cases} 0; & t < 0 \\ 12(1 - e^{-2t}); & 0 \leq t < 3 \leftarrow t=3 = 12(1 - e^{-1 \cdot 3}) \\ 12(e^6 - 1) e^{-2t}; & t \geq 3 \end{cases}$$

$$12(e^6 - 1) e^{-6}$$



Testa själv att beräkna $y_{zs}(t) = \int_{-\infty}^{\infty} x(t-\tau) h(\tau) d\tau$!



b) Beräkna systemets stegsvar $g(t)$

$$g(t) = \mathcal{F}\{u(t)\} = (u * h)(t) = \int_0^t u(t-\tau) h(\tau) d\tau$$

= 0 för $\tau < 0$
: $u(t)=0$ för $t < 0$
 $\Rightarrow u(t-\tau)=0$ för $t-\tau < 0$
dvs. för $\tau > t$

$$= \int_0^t 1 \cdot h(\tau) d\tau = \dots = 3(1 - e^{-2t})u(t)$$

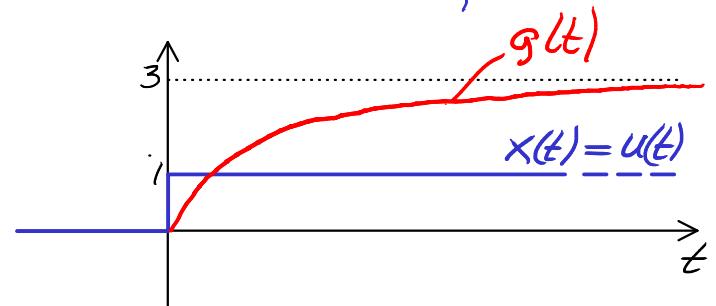
$u(t-\tau)$



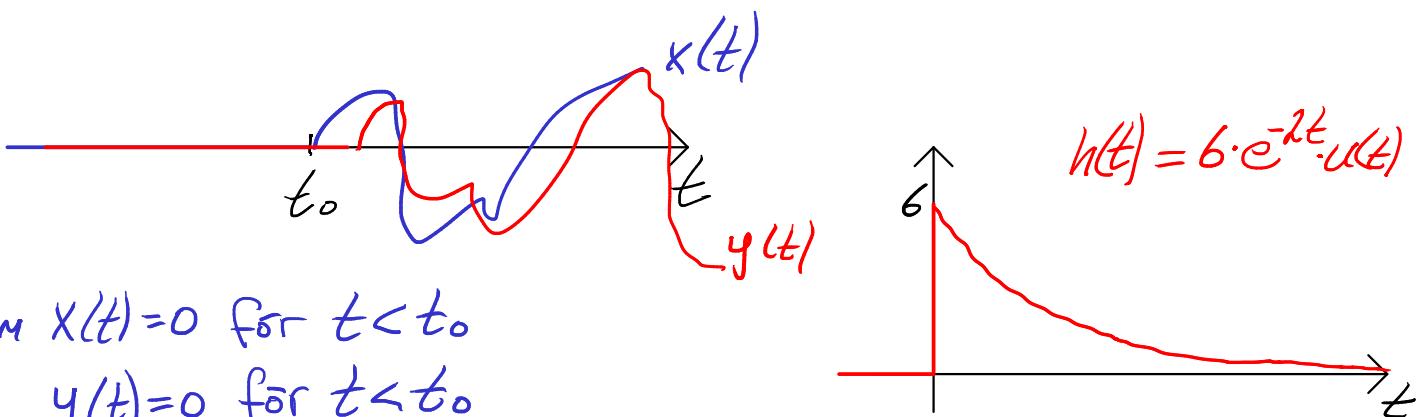
Allmänt:

$$g(t) = \int_{-\infty}^t h(\tau) d\tau$$

$$h(t) = \frac{dg(t)}{dt}$$



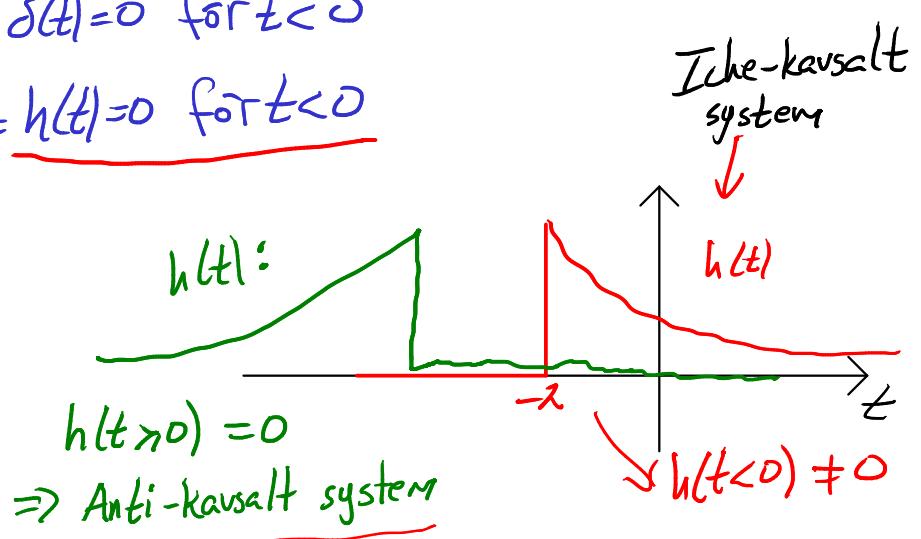
c) Bestäm systemets kausalitetsegenskap



Kausal system $\Rightarrow y(t) = \underline{h(t) = 0 \text{ för } t < 0}$

Här: $h(t < 0) = 0 \Rightarrow$

Kausal system



$$y(t) = \int_{-\infty}^{\infty} x(\tau) h(t-\tau) d\tau$$

$$y(0) = \int_{-\infty}^{0+} x(\tau) h(0-\tau) d\tau + \int_{0+}^{\infty} x(\tau) h(0-\tau) d\tau$$

$$= 0 \text{ för kausal system}$$

$h(-\tau) = 0 \text{ för } \tau > 0$

$\Rightarrow h(\tau) = 0 \text{ för } \tau < 0$

d) Bestäm systemets stabilitetsegenskap

Stabilitet – för LTI-system

Stabilt system:

Varje begränsad insignal ger upphov till en begränsad utsignal.

\Leftrightarrow

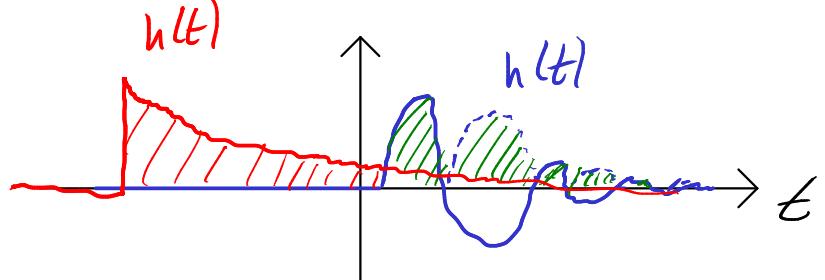
$$\int_{-\infty}^{\infty} |h(t)| dt < \infty$$

$$|y_{zs}(t)| = \left| \int_{-\infty}^{\infty} x(t-\tau) h(\tau) d\tau \right| \leq \int_{-\infty}^{\infty} |x(t-\tau)| |h(\tau)| d\tau \leq M \int_{-\infty}^{\infty} |h(\tau)| d\tau \leq N < \infty$$

$\leq M < \infty$

$$\Rightarrow \int_{-\infty}^{\infty} |h(t)| dt < \infty$$

Exempel på impulssvar $h(t)$ för stabilt system:

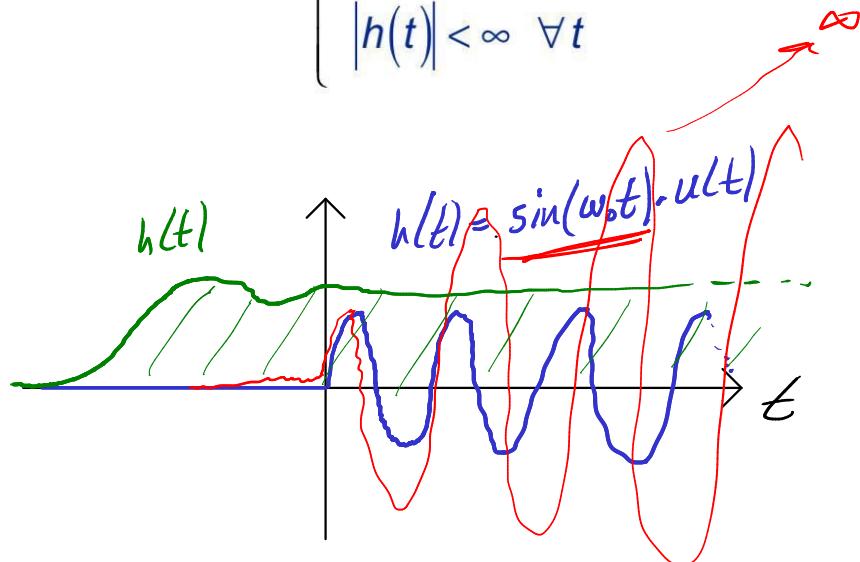


Marginellt stabilt system:

De flesta begränsade insignaler ger upphov till begränsade utsignaler.

$$\Leftrightarrow \begin{cases} \int_{-\infty}^{\infty} |h(t)| dt < \infty \\ |h(t)| < \infty \quad \forall t \end{cases}$$

Exempel på impulssvar $h(t)$ för marginellt stabilt system:

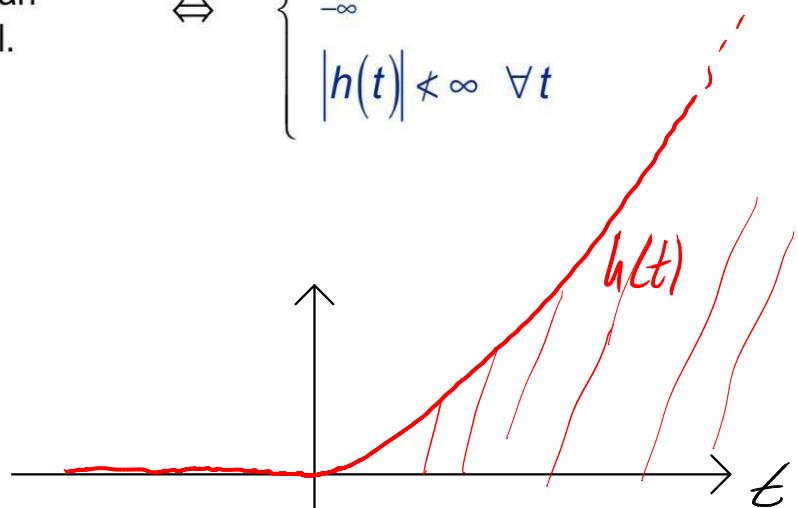


Instabilt system:

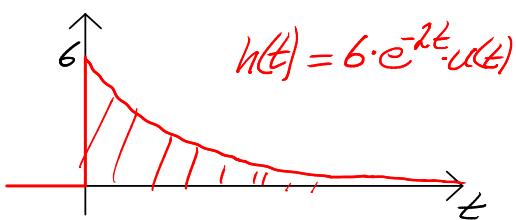
Ingen begränsad nollskild insignal kan ge upphov till en begränsad utsignal.

$$\Leftrightarrow \begin{cases} \int_{-\infty}^{\infty} |h(t)| dt < \infty \\ |h(t)| < \infty \quad \forall t \end{cases}$$

Exempel på impulssvar $h(t)$ för instabilt system:



d) Stabilitetsegenskap för LTI-systemet i räkneuppgiften?



$$\begin{aligned} \int_{-\infty}^{\infty} |h(t)| dt &= \int_0^{\infty} 6 \cdot e^{-2t} dt \\ &= 6 \left[\frac{e^{-2t}}{-2} \right]_0^{\infty} = -3 (e^{-\infty} - e^0) = 3 \end{aligned}$$

\Rightarrow Stabil LTI-system