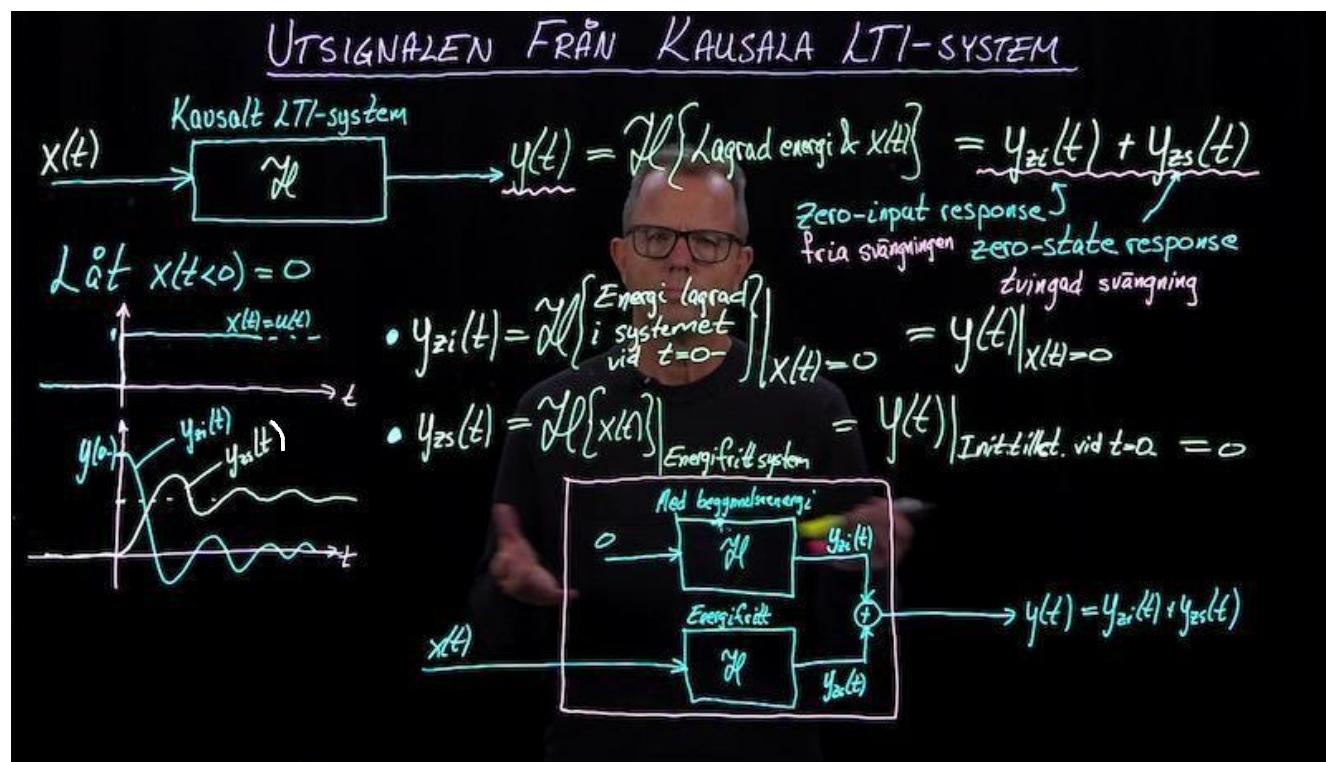


Signaler & System – Föreläsning 2: Tidsdomänanalys av tidskontinuerliga LTI-system

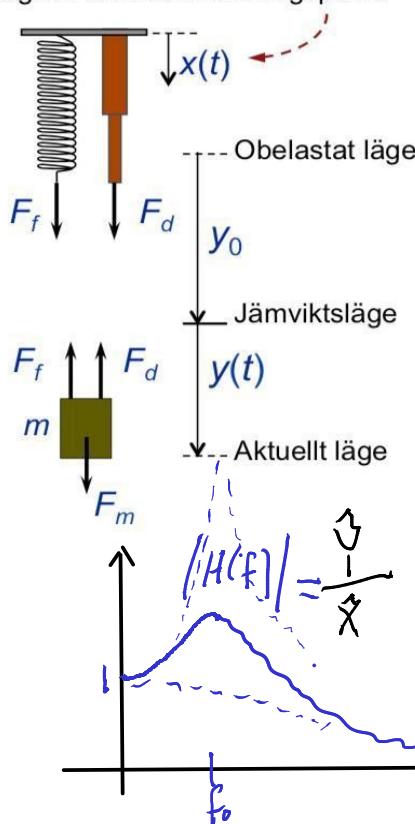
VIDEO 1



Systemexempel 1 – Mekaniskt svängningssystem, massa i dämpad fjäder

Svängande dämpad fjäder – frilägg och sätt ut krafter:

Insignal: ändrad infästningspunkt



$$\text{Fjäderkraften } F_f = k \cdot y_{\text{tot}}(t) = k \cdot (y_0 + y(t) - x(t))$$

$$\text{Dämpkraften } F_d = c \cdot (y_{\text{tot}}(t))' = c \cdot (y'(t) - x'(t))$$

$$\text{Tyngdkraften } F_m = m \cdot g \quad (g = \text{tyngdaccelerationen})$$

$$\text{Newtons 2:a lag: } F_m - F_f - F_d = m \cdot y''(t)$$

$$\Rightarrow m \cdot y''(t) + c \cdot y'(t) + k \cdot y(t) = m \cdot g - k \cdot y_0 + c \cdot x'(t) + k \cdot x(t)$$

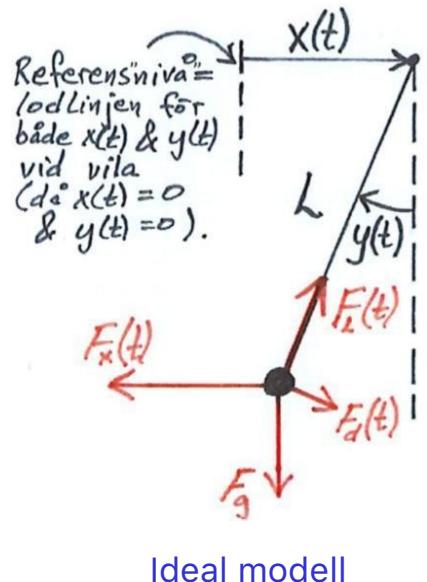
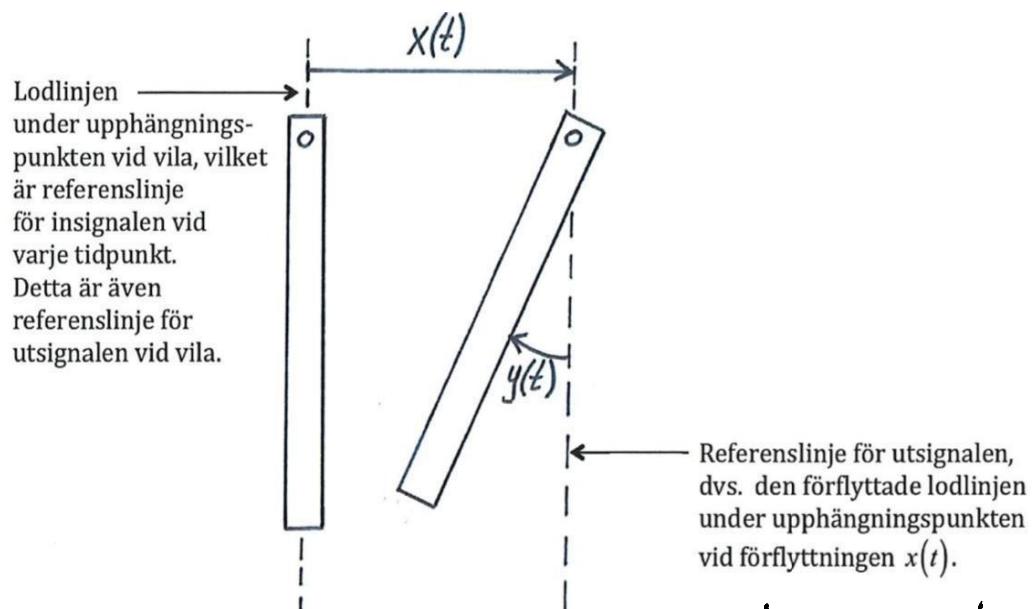
$$\text{Vid vila är } x=0, x'=0, y=0, y'=0, y''=0 \Rightarrow m \cdot g = k \cdot y_0$$

\Rightarrow

$$m \frac{d^2 y(t)}{dt^2} + c \frac{dy(t)}{dt} + k \cdot y(t) = c \frac{dx(t)}{dt} + k \cdot x(t)$$

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Systemexempel 2 – Mekaniskt svängningssystem, pendlande linjal



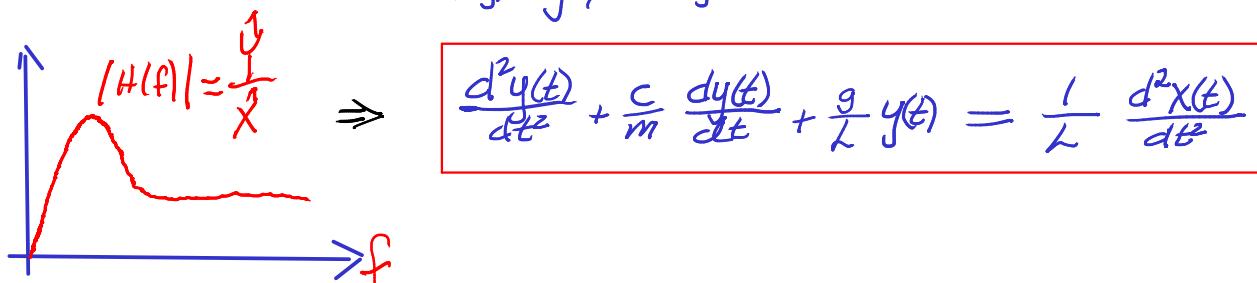
Newton's 2:a lag \Rightarrow

$$y'' + \frac{c}{m} y' + \frac{g}{m} \sin(y) = \frac{\cos(y)}{L} \cdot x''$$

Icke-linjärt system, p.g.a. $\sin(y)$ och $\cos(y)$

\Rightarrow Linjärisera, dvs. approximera med linjär modell:

Om vinkeln y är liten $\Rightarrow \sin(y) \approx y, \cos(y) \approx 1$



DEN FRIA SVÄNGNINGEN, ZERO-INPUT RESPONSE $y_{zi}(t)$

$x(t)=0$



Differentialekvationsbeskrivning:

$$a_N \cdot \frac{d^N y(t)}{dt^N} + a_{N-1} \cdot \frac{d^{N-1} y(t)}{dt^{N-1}} + \dots + a_1 \cdot \frac{dy(t)}{dt} + a_0 \cdot y(t) = b_M \cdot \frac{d^M x(t)}{dt^M} + b_{M-1} \cdot \frac{d^{M-1} x(t)}{dt^{M-1}} + \dots + b_1 \cdot \frac{dx(t)}{dt} + b_0 \cdot x(t)$$

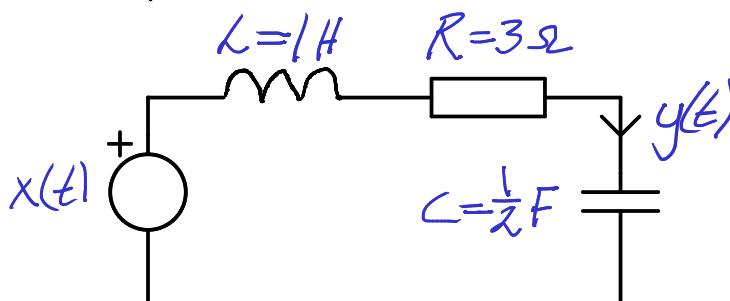
(Vanligt: $N > M$)

Deriveringsoperatorn \mathcal{D} : $\mathcal{D} y(t) = \frac{dy(t)}{dt}$, $\mathcal{D}^i y(t) = \frac{d^i y(t)}{dt^i}$

$$\Rightarrow Q(\mathcal{D})y(t) = P(\mathcal{D})x(t) \quad \text{där} \quad \begin{cases} Q(\mathcal{D}) = a_N \mathcal{D}^N + a_{N-1} \mathcal{D}^{N-1} + \dots + a_1 \mathcal{D} + a_0 \\ P(\mathcal{D}) = b_M \mathcal{D}^M + b_{M-1} \mathcal{D}^{M-1} + \dots + b_1 \mathcal{D} + b_0 \end{cases}$$

$$\Rightarrow \text{lös } Q(\mathcal{D})y_{zi}(t) = 0 \quad \Rightarrow \quad y_{zi}(t) = \sum \text{(karakteristiska termer)}$$

e^{kt} , $t^n e^{kt}$, $e^{at} \cos(\beta t)$

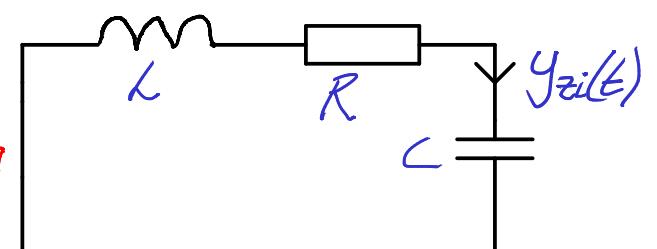
OBS: $y_{zi}(t)$ betecknas $y_0(t)$ i kursboken!Exempel:

$$\frac{d^2 y(t)}{dt^2} + 3 \frac{dy(t)}{dt} + 2 y(t) = \frac{dx(t)}{dt}$$

$$y(0) = 0, \quad y'(0) = -5$$

Beräkna $y_{zi}(t)$

Ekvivalent krets:



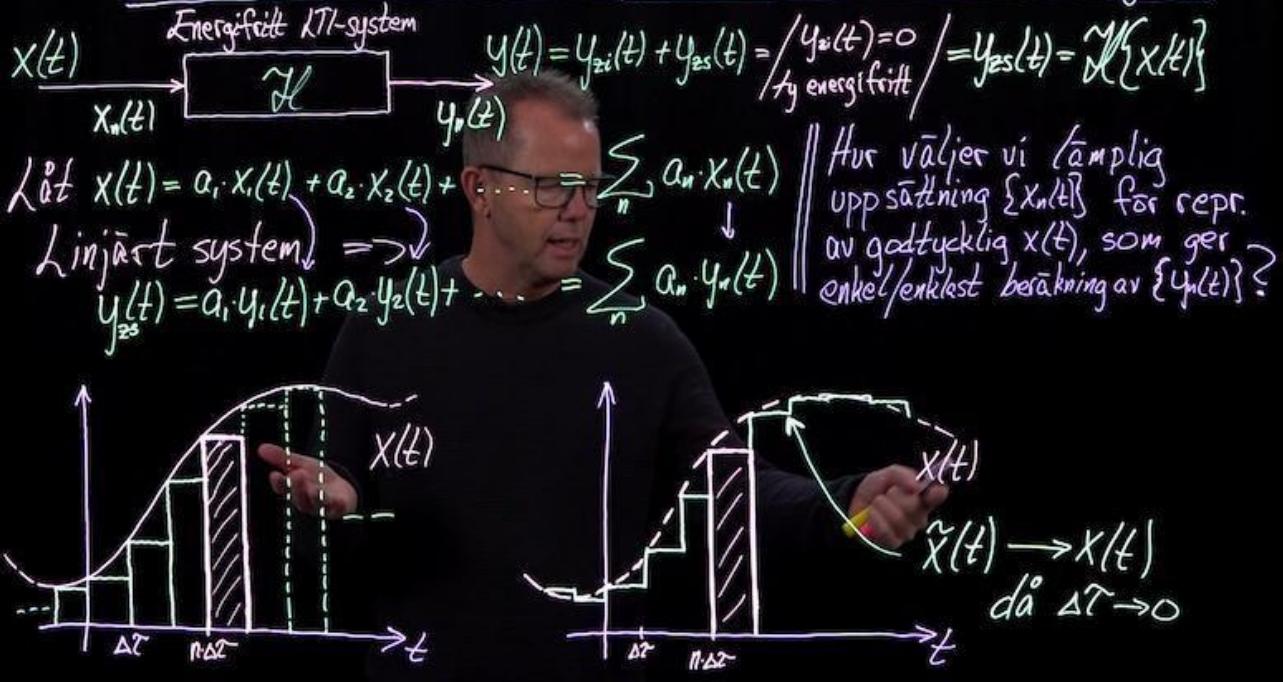
$$(\mathcal{D}^2 + 3\mathcal{D} + 2)y_{zi}(t) = 0$$

$$\Rightarrow \text{Kar. ekv. } \lambda^2 + 3\lambda + 2 = 0$$

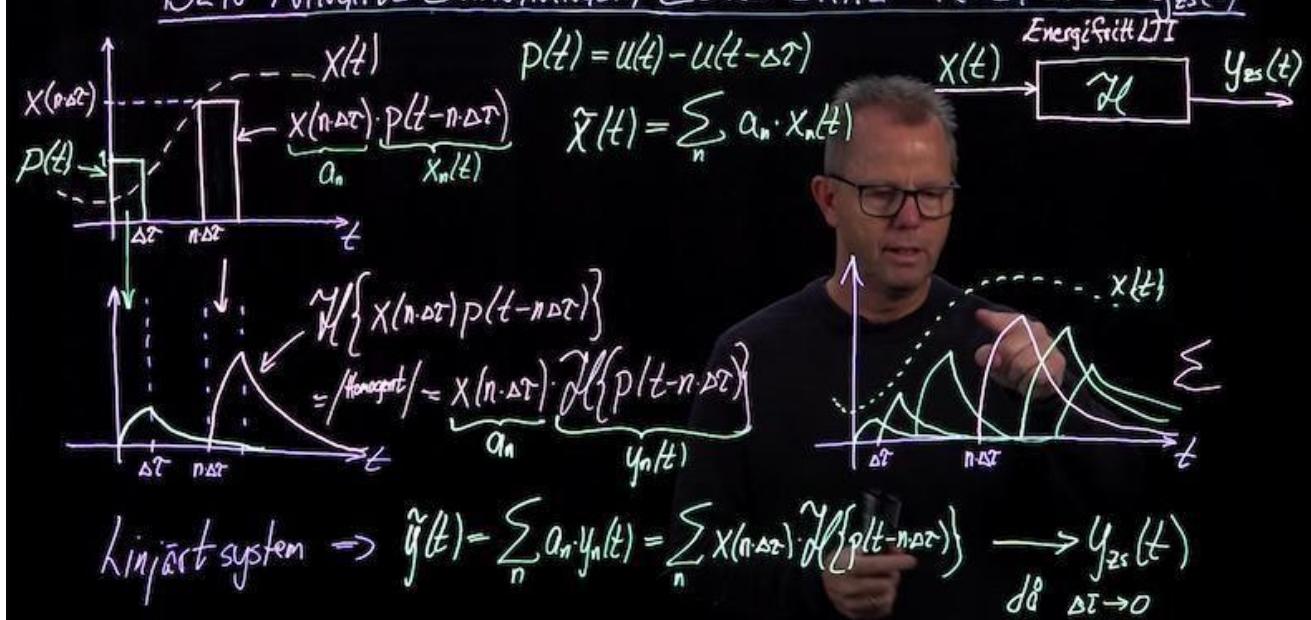
$$\Rightarrow \begin{cases} \lambda_1 = -1 \\ \lambda_2 = -2 \end{cases} \Rightarrow y_{zi}(t) = K_1 e^{-t} + K_2 e^{-2t} = \dots$$

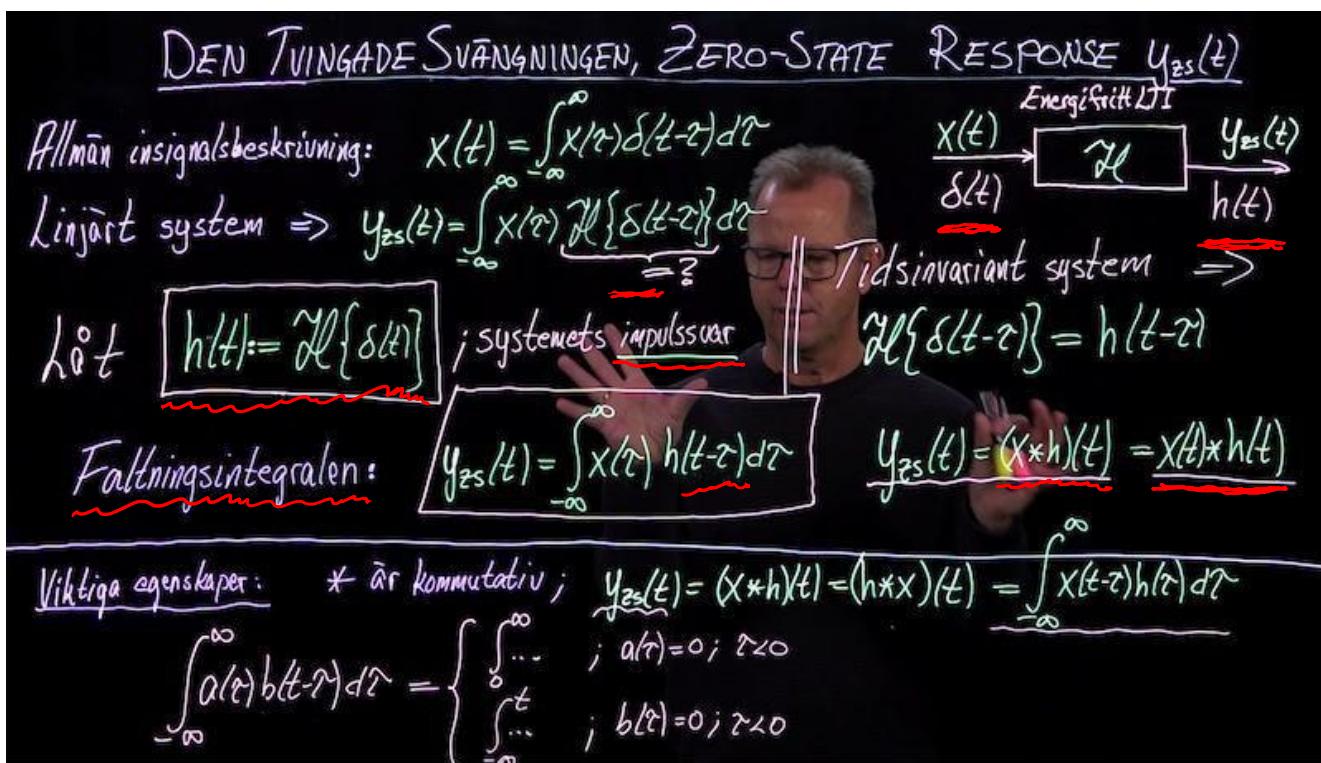
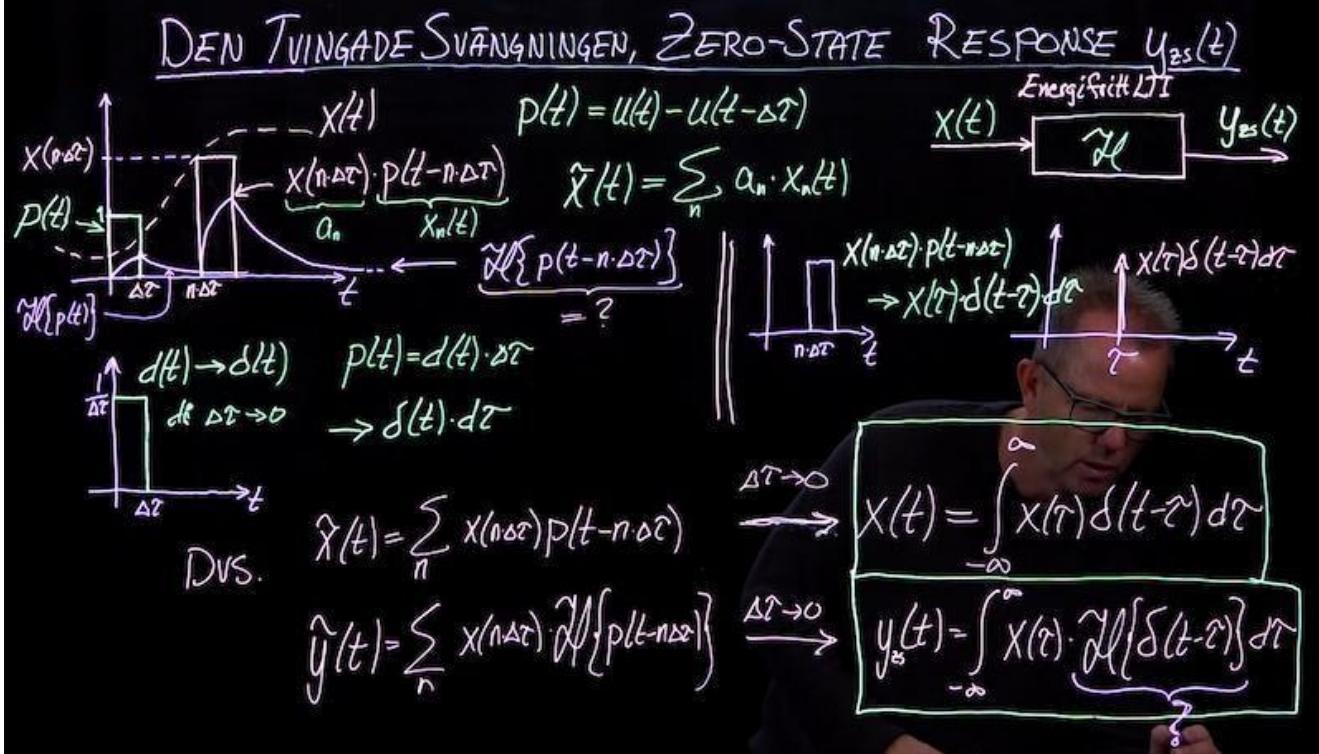
$$= -5e^{-t} + 5e^{-2t}; \quad t \geq 0$$

DEN Tvingade Svängningen, ZERO-STATE RESPONSE $y_{zs}(t)$



DEN Tvingade Svängningen, ZERO-STATE RESPONSE $y_{zs}(t)$





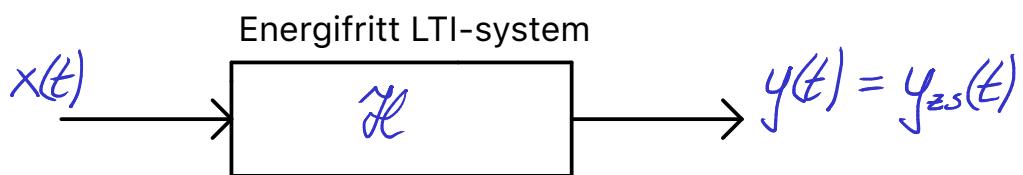
Notera: Om vi utgår från $x(t) = \int_{-\infty}^{\infty} x(\tau) \delta(t-\tau) d\tau \Rightarrow$

$$y_{zs}(t) = \mathcal{H}\{x(t)\} = \mathcal{H}\left\{\int_{-\infty}^{\infty} x(\tau) \delta(t-\tau) d\tau\right\} = / \text{linjärt system} /$$

$$= \int_{-\infty}^{\infty} x(\tau) \mathcal{H}\{\delta(t-\tau)\} d\tau = \begin{cases} \mathcal{H}\{\delta(t)\} = h(t) & \& \\ \text{Tidsinvariant system} & \& \\ \Rightarrow \mathcal{H}\{\delta(t-\tau)\} = h(t-\tau) & \& \end{cases} = \int_{-\infty}^{\infty} x(\tau) h(t-\tau) d\tau$$

Slutsats efter videon ovan:

Utsignalens zero-statekomponent (den tvingade svängningen):



där

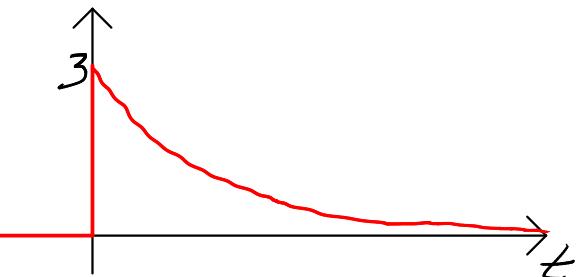
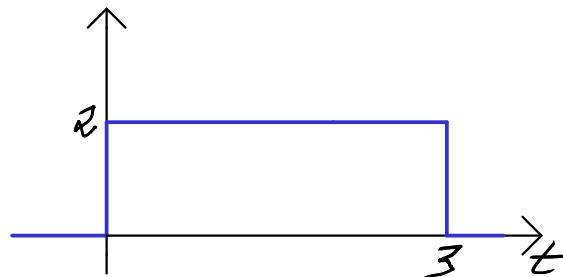
$$y_{zs}(t) = (x * h)(t) = \int_{-\infty}^{\infty} x(\tau)h(t-\tau)d\tau = \int_{-\infty}^{\infty} x(t-\tau)h(\tau)d\tau$$

Eller $x(t) * h(t)$

Faltningsintegralen/-erna

där $\underline{h(t) = \delta(t)}$ är LTI-systemets impulssvar

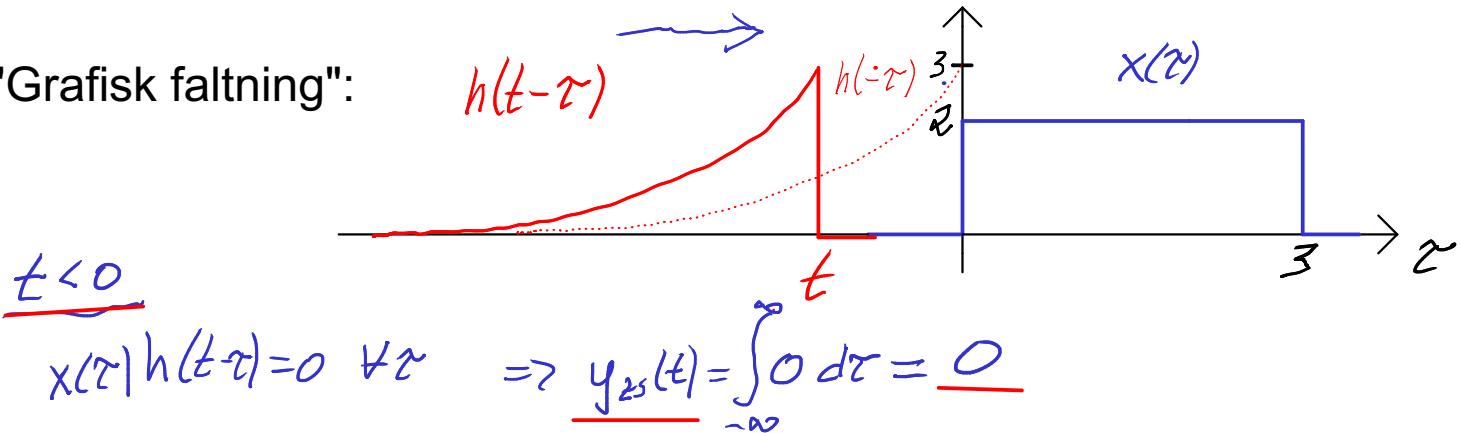
Exempel:



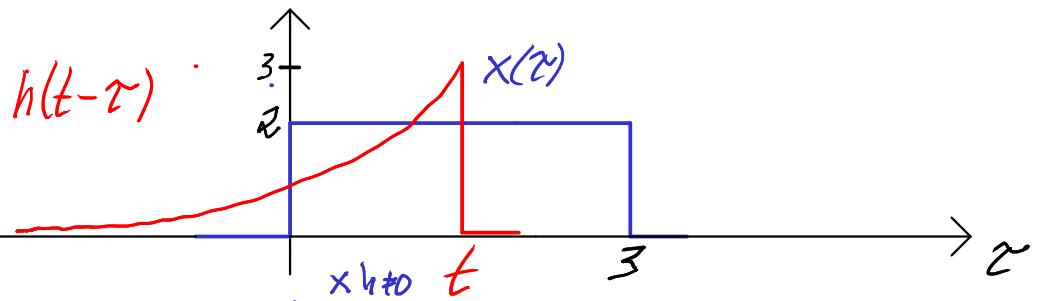
ordn. 1

$$y_{zs}(t) = (x * h)(t) = \int_{-\infty}^{\infty} x(\tau)h(t-\tau)d\tau$$

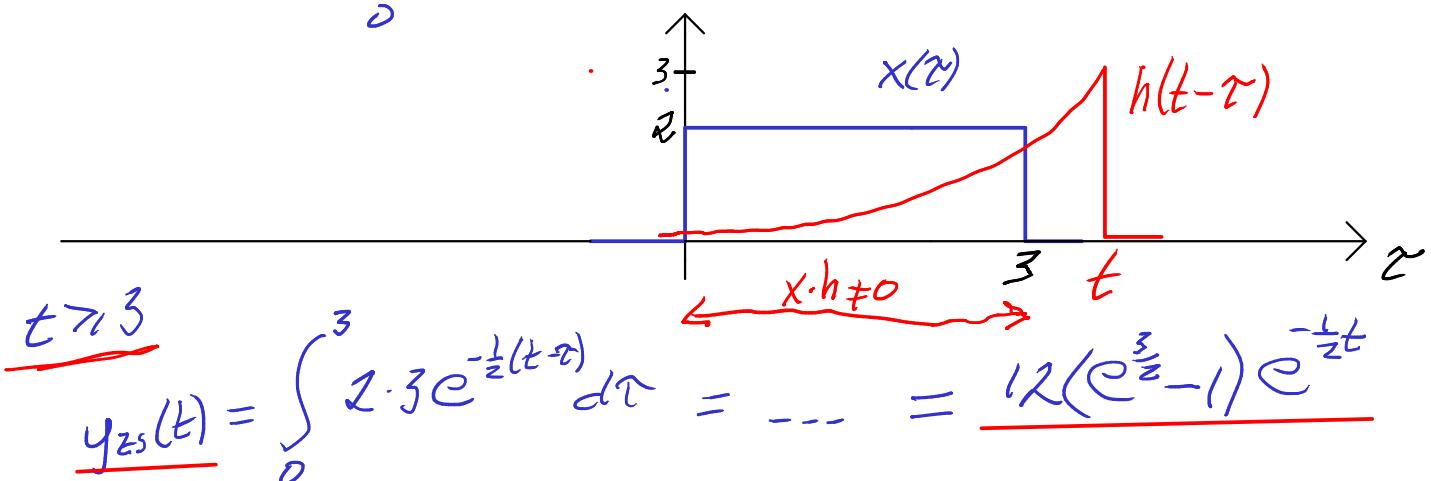
"Grafisk faltung":



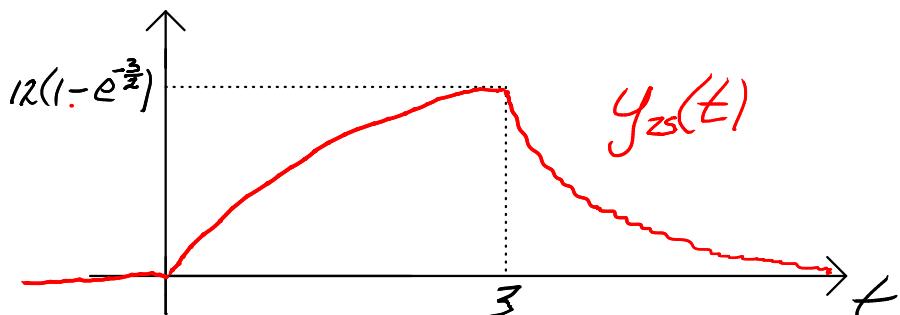
$$0 \leq t < 3$$



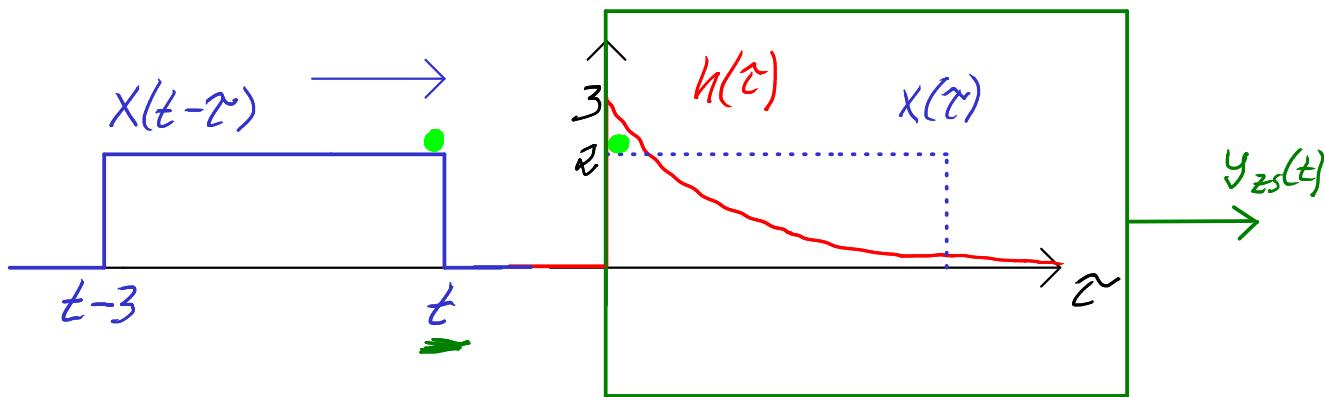
$$\begin{aligned} y_{zs}(t) &= \int_{-\infty}^0 0 dt + \int_0^t 2 \cdot 3 e^{-\frac{1}{2}(t-\tau)} d\tau + \int_t^\infty 0 dt \\ &= 6 e^{-\frac{1}{2}t} \int_0^t e^{\frac{1}{2}\tau} d\tau = \dots = 12(1 - e^{-\frac{1}{2}t}) \end{aligned}$$



$$\therefore y_{zs}(t) = \begin{cases} 0; & t < 0 \\ 12(1 - e^{-\frac{1}{2}t}); & 0 \leq t < 3 \\ 12(e^{\frac{3}{2}} - 1)e^{-\frac{1}{2}t}; & t \geq 3 \end{cases}$$



Testa gärna själv att beräkna $y_{zs}(t) = \int_{-\infty}^{\infty} x(t-\tau) h(\tau) d\tau$!



Impulssvaret $h(t)$ och kausalitet

$$\text{Om } x(t < t_0) = 0 \Rightarrow y_{zs}(t < t_0) = 0$$

Kausal system

Ex:

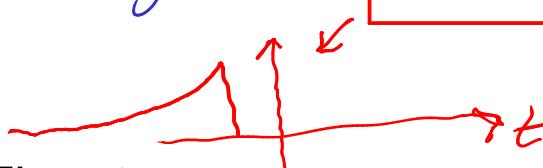
$$x(t < 0) = 0 \Rightarrow h(t < 0) = 0$$

h(t < 0) = 0

$h_2(t) = 3e^{-\frac{1}{2}(t+2)} u(t+2)$ $h(t) = 3e^{-\frac{1}{2}t} u(t)$

Icke-kausal

Anti-kausal system: h(t) = 0 ; t > 0



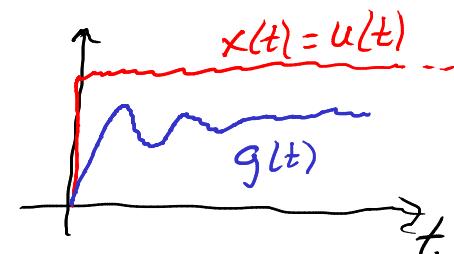
Stegsvar för LTI-system

$$\underline{g(t)} = \mathcal{H}\{u(t)\} = (u * h)(t) = \int_{-\infty}^{\infty} \underline{u(t-\tau)} \underline{h(\tau)} d\tau =$$

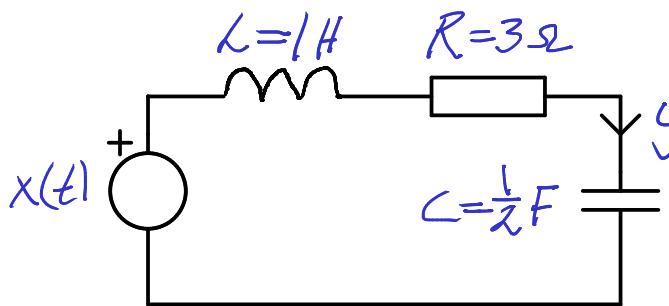
$\int_{-\infty}^{\infty} \underline{h(\tau)} d\tau ; t-\tau < 0 \Rightarrow \tau > t$

$$= \int_{-\infty}^t h(\tau) d\tau$$

$$\Rightarrow h(t) = \frac{dg(t)}{dt}$$



Tidigare exempel:

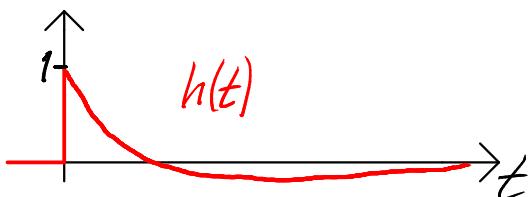


$$\frac{d^2y(t)}{dt^2} + 3\frac{dy(t)}{dt} + 2y(t) = \frac{dx(t)}{dt}$$

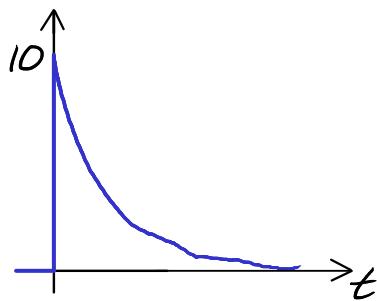
$$y(0) = 0, \quad y'(0) = -5$$

$$\Rightarrow \underline{y_{zi}(t) = -5e^{-t} + 5e^{-2t}; \quad t \geq 0}$$

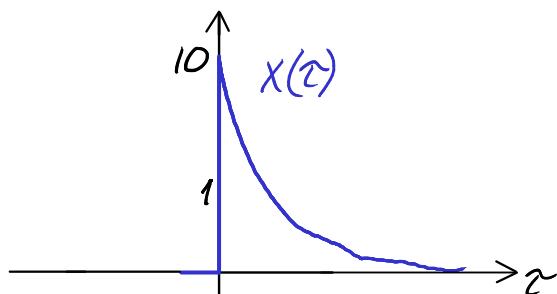
Man kan visa att detta LTI-system
har impulssvar $h(t) = (2e^{-2t} - e^{-t})u(t)$



Låt insignalen vara
 $x(t) = 10e^{-3t}u(t)$



Beräkna den totala utsignalen $y(t) = y_{zi}(t) + y_{zs}(t)$



$$y_{zs}(t) = \int_{-\infty}^{\infty} x(\tau) h(t-\tau) d\tau$$